

Pipe Flow

Fully Developed Laminar Flow:

Dimensionless analysis

$$\Delta p = f(\bar{v}, \ell, D, \mu) \rightarrow \frac{D\Delta p}{\mu\bar{v}} = \phi\left(\frac{\ell}{D}\right) \quad \text{Eq (8.17)}$$

Assumption: $\Delta p \sim \ell$

$$\frac{D\Delta p}{\mu\bar{v}} = C'\left(\frac{\ell}{D}\right)$$

$$Q = A\bar{v} = \frac{\pi D^4}{4C'\mu\ell} \Delta p \quad \text{Eq (8.18)}$$

The constant $C' = 32$ for a round pipe

Pipe Flow

Fully Developed Laminar Flow:

Dimensionless analysis

$$\Delta p = C' \left(\frac{\ell}{D} \right) \frac{\mu \bar{v}}{D}$$

Making all dimensionless:

$$\frac{\Delta p}{\frac{1}{2} \rho \bar{v}^2} = \frac{2C'}{\text{Re}} \left(\frac{\ell}{D} \right) = f_D \left(\frac{\ell}{D} \right)$$

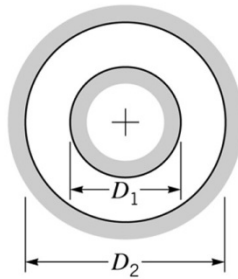
f_D is the Darcy friction factor

Pipe Flow

Fully Developed Laminar Flow:

I. Concentric Annulus

$$D_h = D_2 - D_1$$



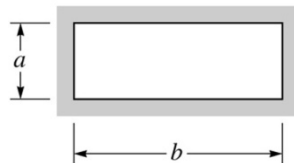
$$f_D = \frac{2C'}{\text{Re}} = \frac{C}{\text{Re}}$$

$C = 64$ for a round pipe

For noncircular ducts:

II. Rectangle

$$D_h = \frac{2ab}{a+b}$$



$$f_D = \frac{C}{\text{Re}_h}$$

where $\text{Re}_h = \frac{\rho \bar{v} D_h}{\mu}$

$$D_h = \frac{4A}{P}$$

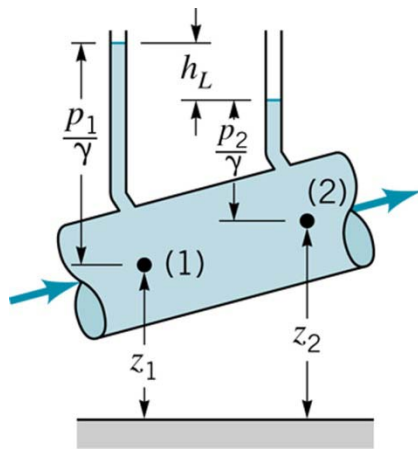
$A = \text{area}$; $P = \text{perimeter}$
 $D_h = \text{hydraulic diameter}$

Pipe Flow

Fully Developed Laminar Flow:

Energy consideration

$$\frac{\Delta p}{\gamma} + \frac{\Delta \alpha \bar{v}^2}{2g} + \Delta z = h_L$$



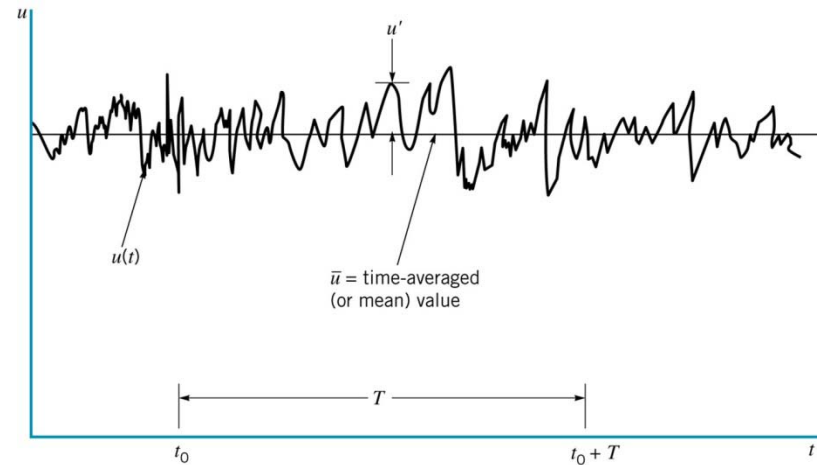
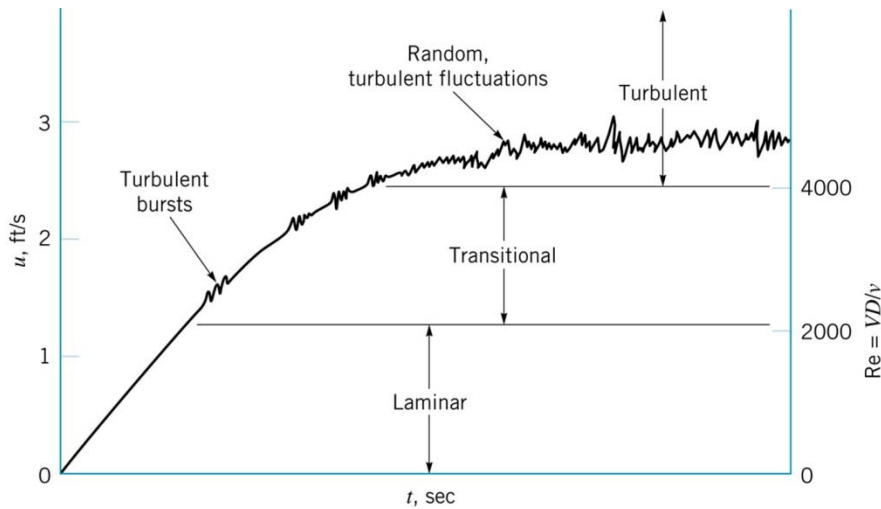
Fully developed flow: $\frac{\Delta \alpha \bar{v}^2}{2g} = 0$

$$\left. \begin{aligned} \frac{\Delta p + \gamma \Delta z}{\ell} &= \frac{4\tau_w}{D} \\ \frac{\Delta p}{\gamma} + \Delta z &= h_L \end{aligned} \right\} \rightarrow h_L = \frac{4\tau_w \ell}{\gamma D}$$

Eq (8.23)

Pipe Flow

Transition from Laminar to Turbulent Flow:



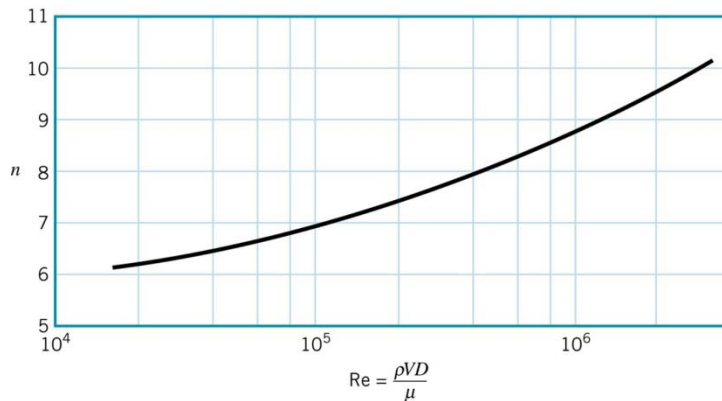
Pipe Flow

Fully Developed Turbulent Flow:

Power-law velocity profile:

$$\bar{u}(r) = v_c \left[1 - \left(\frac{r}{R} \right) \right]^{\frac{1}{n}}$$

Eq (8.31)



Approximation for practical flow: $n = 7$

